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# UNITED STATES EXPERIMENTAL MODEL BASIN

NAVY YARD, WASHINGTON, D.C.

## TEST OF TRANSOM STERNS ON DESTROYERS

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EXPERIMENTAL MODEL BASIN  
ERECTED 1898  
BUREAU OF  
CONSTRUCTION AND REPAIR  
NAVY DEPARTMENT

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NOVEMBER 1932

REPORT NO. 339



**TEST of TRANSOM STERNS on DESTROYERS**

**U. S. Experimental Model Basin  
Navy Yard, Washington, D. C.**

**November, 1932**

**Report No. 339.**



## TEST of TRANSOM STERNS on DESTROYERS

### Summary

A series of nine 20-ft. models of destroyers was made in which the bows were identical, the dimensions, including displacement, were constant, and the stern load water lines were practically alike. The models differed by variations in the immersion of their transom sterns and in the shape of their buttock lines.

The models were towed for resistance at the designed trim and at other conditions of trim and the curves of resistance plotted on speed.

The resistances of the models at selected speeds were plotted on variations in transom immersion and shape of sectional area curves. Contour curves of resistance were used to prepare diagrams showing the influence of the variations on resistance.

### Introduction

The transom stern has been embodied in the design of destroyers for many years and a systematic test of some of its characteristics was considered desirable. Of these characteristics the depth of immersion of the transom and the shape of the buttock lines as they approach the transom were chosen for this investigation.

Of previous destroyer models, No. 3092 had given the best results as regards resistance so it was decided to use the bow of this model with nine different sterns, the variations in which were restricted as far as possible to the features under investigation.

### Methods and Apparatus

For this test nine 20-ft. models were made. The bows of these models, from the midship section forward, were identical, copied from the Model Basin lines of a destroyer for the FARRAGUT class tested as model No. 3092. The bow sectional area curve is shown in Fig. 2.

Aft of the midship section (which is 60 per cent of the l.w.l. length from the F.P.) the general shape of the stern was made as much like the lines of model No. 3092 as the desired variations would allow. The length, displacement, drag of keel, lowest point of keel and the load water line were held constant except that when the transom had no immersion the l.w.l. was necessarily cut away near the transom, but the knuckle immediately above continued the curve to the transom so that the load water lines when under way were practically the same for all models.

The transom of each model was simply a straight vertical cut normal to the center plane of the model.

The depths of immersion chosen for the transoms were such as to give areas

below the l.w.l equal to 0, 10 per cent and 20 per cent of the area of the immersed midship section.

Each of these depths of transom was combined with three sectional area curves as shown on Fig. 1; these curves produced buttock lines of corresponding curvature, hollow, normal or full, as they approached the transom.

Figs. 3 and 4 show the type of sterns used; the bossings shown were added for subsequent tests.

These models were run for resistance up to a speed of 10 knots, corresponding to a speed of 41 knots for a 334-ft. ship. They were without appendages and were tested at three trims, namely, designed water line, 3 in. by bow and 3 in. by stern (for the model).

After drawing the curves of resistance plotted on speed, the resistances were read off at speeds of 6 knots, 7.5 knots, and 9 knots. These values are tabulated on Fig. 5.

For each of the three speeds and the three trims a diagram was made, plotting contours of resistance on values of immersed transom area and fullness of sectional area curve. In these diagrams the abscissae are relative only as the changes in sectional area curves, could not be plotted quantitatively.

These diagrams, Figs. 6 to 14 inclusive, show the effect on resistance of variations in the immersion of transom and in fullness of sectional area curve as it approaches the transom.

As the comparative results of tests of models run bare hull for resistance only do not always agree with those of the same models run self-propelled, two of the models, 3144 and 3143, showing the best and the worst performance respectively, were selected for comparison when self-propelled. The bossings were made similar to those on model 3092 and the same propellers were used. The results are shown on Figs. 15 and 16.

### Discussion of Results

The contour sheets, Figs. 6 to 14, show in a general way the effect of the immersion of the transom and the shape of the buttock lines on resistance.

From these contours it may be seen that as the speed is increased the optimum immersed transom area is increased, and that at speeds corresponding to 25 to 30 knots for the 334-ft. ship, the best area is about 10 per cent of the area of the midship section. The advantage of hollow buttocks is also shown for speeds of 30 knots and above.

A comparison of the self-propelled tests with the resistance tests run in bare hull condition shows a greater relative difference in SHP than EHP. The propulsive coefficient of the best model is higher except at the highest speed, where the propulsive coefficients are about the same. It is considered safe to apply the conclusions drawn from the resistance tests to the self-propelled

tests, and thence to the design of the ships, in so far as such design may be based on model tests.

### Conclusion

For destroyers of the general design and speed investigated, the transom should have an immersed area about 10 per cent of the immersed midship section and the buttock lines immediately forward of the transom should be hollow, especially when the design requires the best performance at the highest speeds.

If economical performance at cruising speeds is to govern, the transom stern should not be considered. At the lower speeds selected for cruising, say 16 knots, there is a disadvantage in submerging the transom, as shown by the table in Fig. 5a.

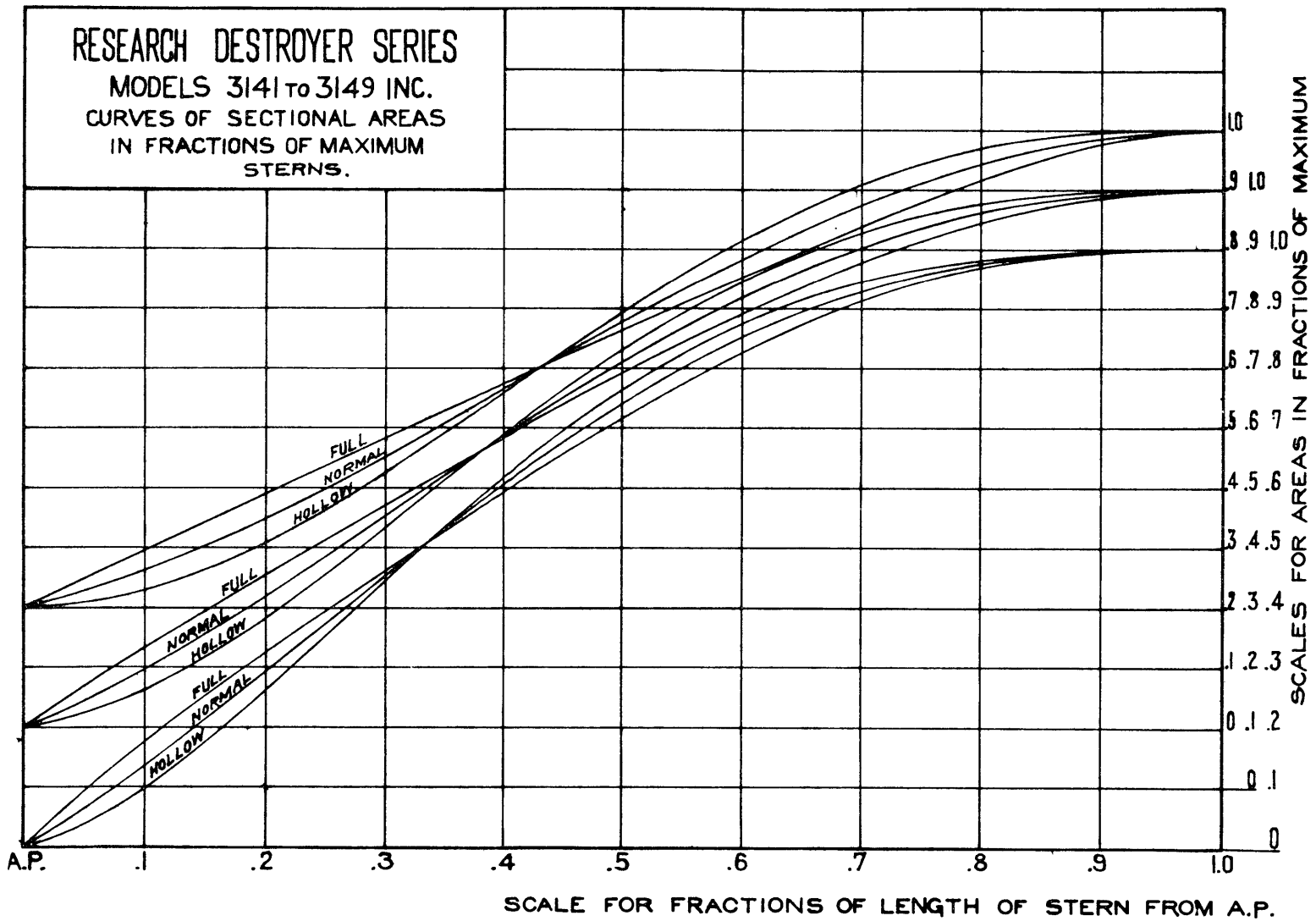
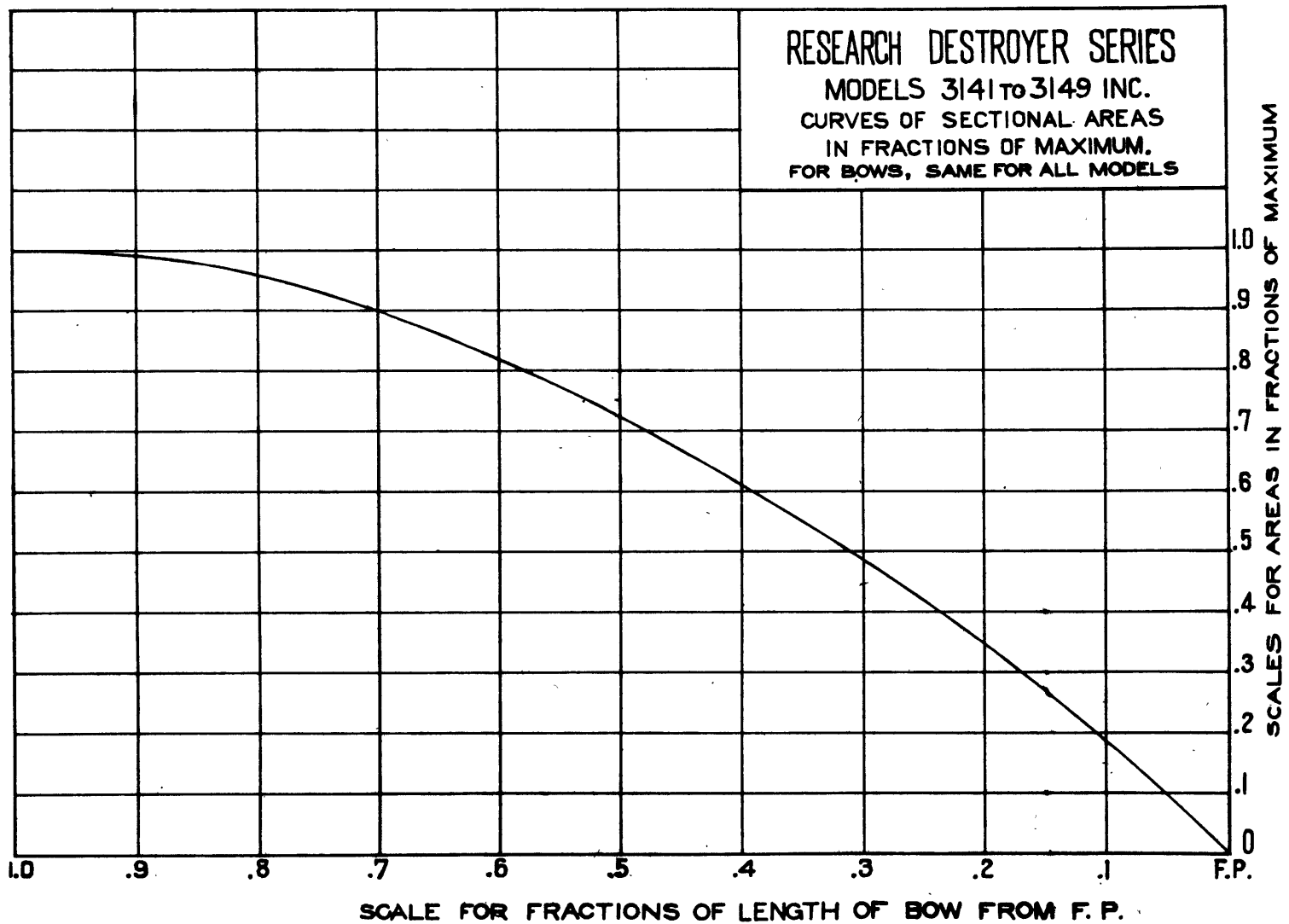
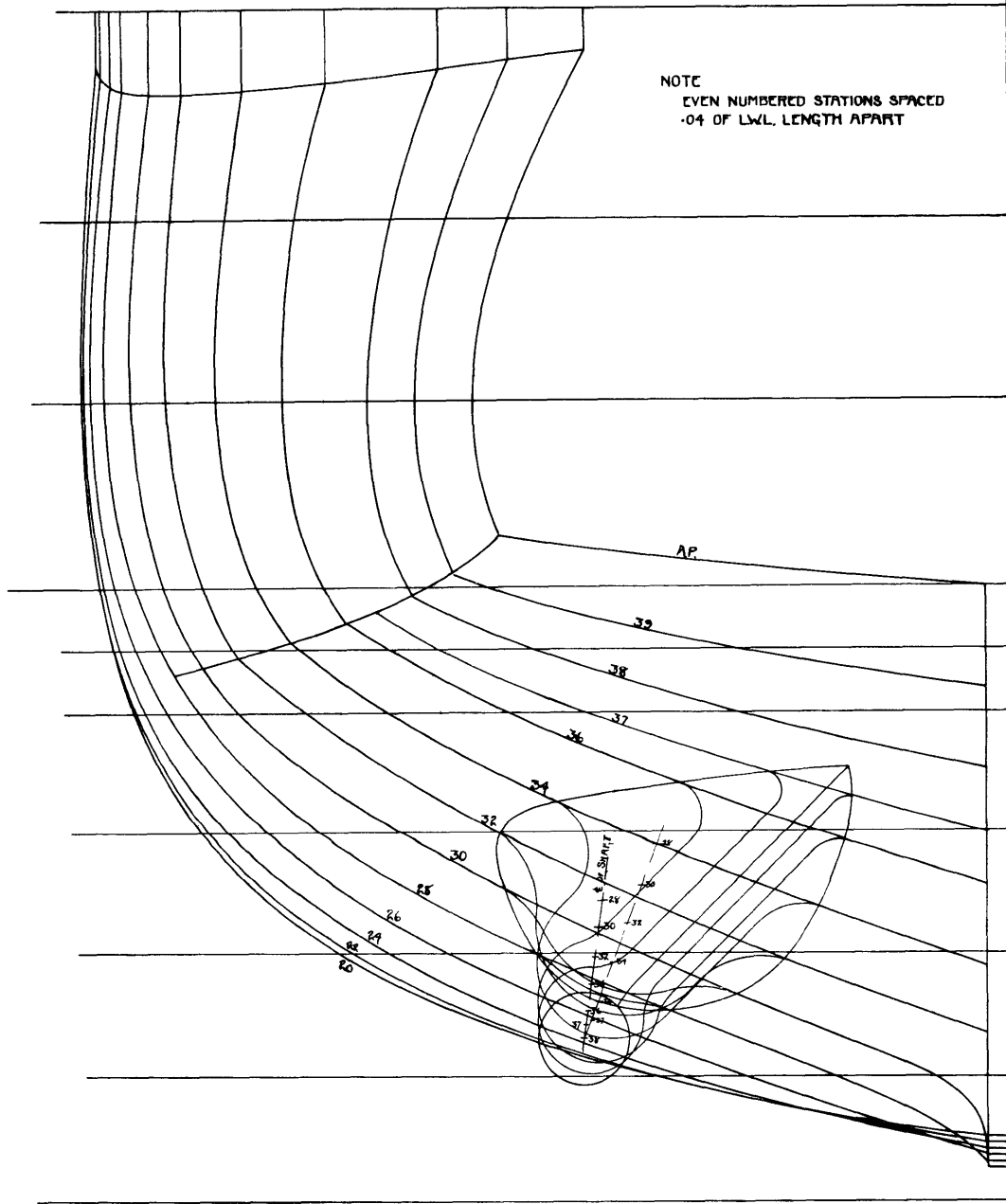


FIG. 1





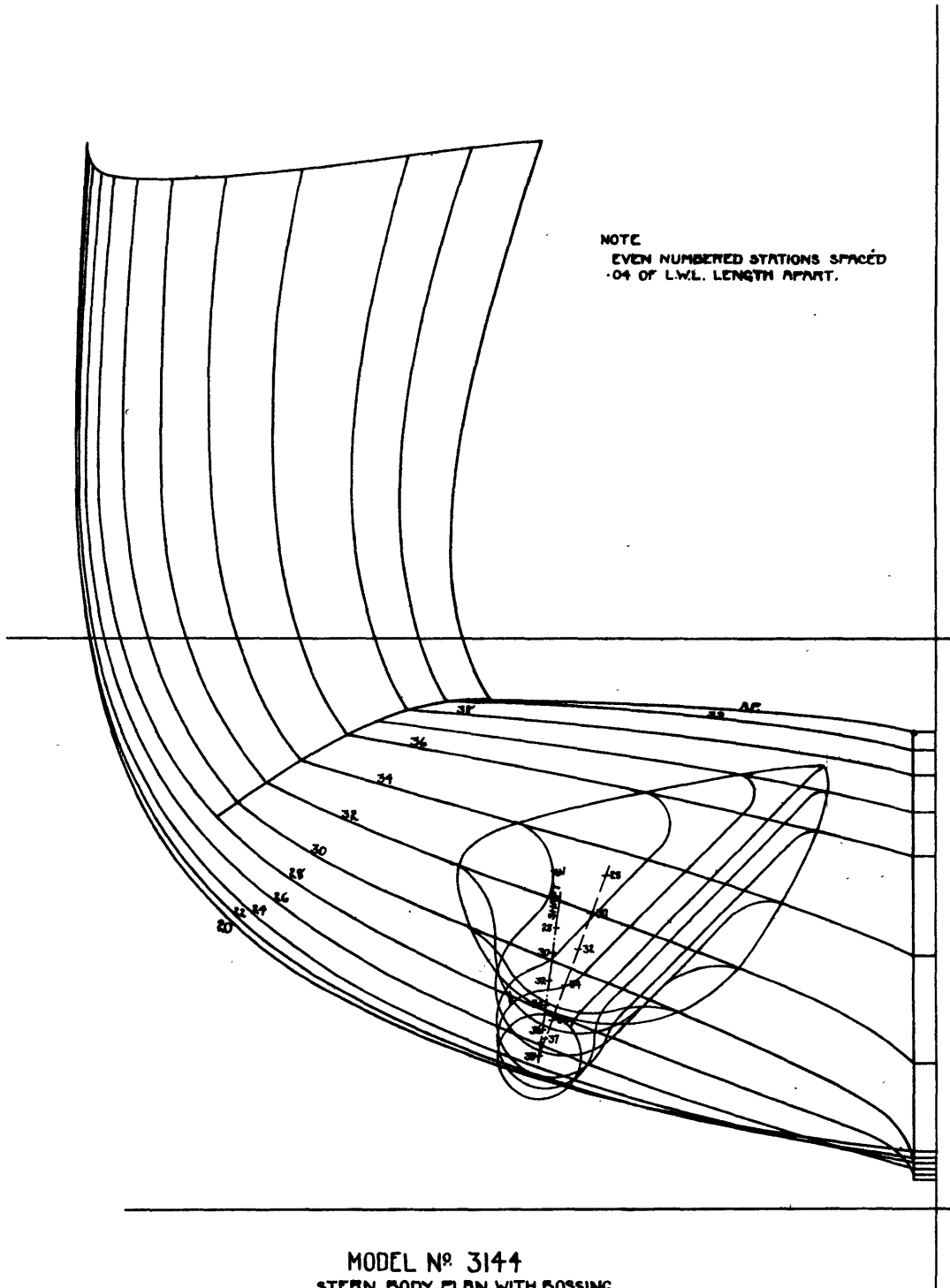
**FIG. 2**



MODEL N° 3143  
STERN BODY PLAN WITH BOSSING

BOSS DIAM TO SUIT PROPS 1182 & 1183  
PROP CLEARANCE SAME AS 3092  
SHAFT SLOPE  
TAPER OF BOSS SIMILAR TO

FIG. 3



MODEL N<sup>o</sup> 3144  
STERN BODY PLAN WITH BOSSING

BOSS DIAM. TO SUIT PROP'S 1182 & 7183  
PROP CLEARANCE SAME AS 3092  
SHAFT SLOPE " " " "  
TAPER OF BOSS SIMILAR TO " "

FIG. 4.

RESEARCH DESTROYER SERIES

MODELS 3141 TO 3149 INC.

DISPLACEMENT 810 LBS

VALUES OF TOTAL RESISTANCE AT VARIOUS SPEEDS

MEAN TEMP. OF WATER = 56°

T = IMMERSSED AREA OF TRANSOM  
AT ZERO TRIM

SAME BOW FOR ALL MODELS

A = AREA OF MIDSHIP SECTION

HOLLOW, NORMAL, FULL, STERNS

			V = 6.0 KNOTS.			V = 7.5 KNOTS.			V = 9.0 KNOTS		
MODEL NO.	CHARACTER OF STERN	$\frac{T}{A}$	RESISTANCE IN LBS.			RESISTANCE IN LBS			RESISTANCE IN LBS.		
			DESIGNED WL.	3" BY BOW	3" BY STERN	DESIGNED WL.	3" BY BOW	3" BY STERN	DESIGNED WL.	3" BY BOW	3" BY STERN
3141	HOLLOW	0	19.85	20.30	21.60	36.30	37.00	37.80	50.35	50.90	50.60
3142	NORMAL	"	20.15	20.50	21.65	36.75	37.60	37.95	49.95	50.95	50.20
3143	FULL	"	21.50	21.15	23.45	39.00	38.70	41.30	52.95	52.50	53.95
3144	HOLLOW	.10	19.30	19.55	20.75	34.60	35.30	37.00	46.55	47.70	48.50
3145	NORMAL	"	18.00	18.15	20.80	34.80	36.05	36.70	47.10	49.50	47.70
3146	FULL	"	20.10	19.75	23.00	36.00	36.30	38.20	48.75	49.90	50.20
3147	HOLLOW	.20	20.40	19.80	22.75	35.70	35.50	37.50	47.00	48.40	48.00
3148	NORMAL	"	20.25	19.80	23.15	35.60	36.10	37.80	48.25	49.25	49.30
3149	FULL	"	20.55	19.99	22.90	35.90	35.40	37.75	47.40	48.20	48.65

FIG. 5

## RESEARCH DESTROYER SERIES

MODELS 3141 TO 3149 INC.

DISPLACEMENT 810 LBS.

VALUES OF TOTAL RESISTANCE AT CRUISING SPEEDS

MEAN TEMP. OF WATER = 56°

2.94 KN. ⇄ 12 KN. & 3.91 KN. ⇄ 16 KN. FOR 334' SHIP

T = IMMERSSED AREA OF TRANSOM

SAME BOW FOR ALL MODELS

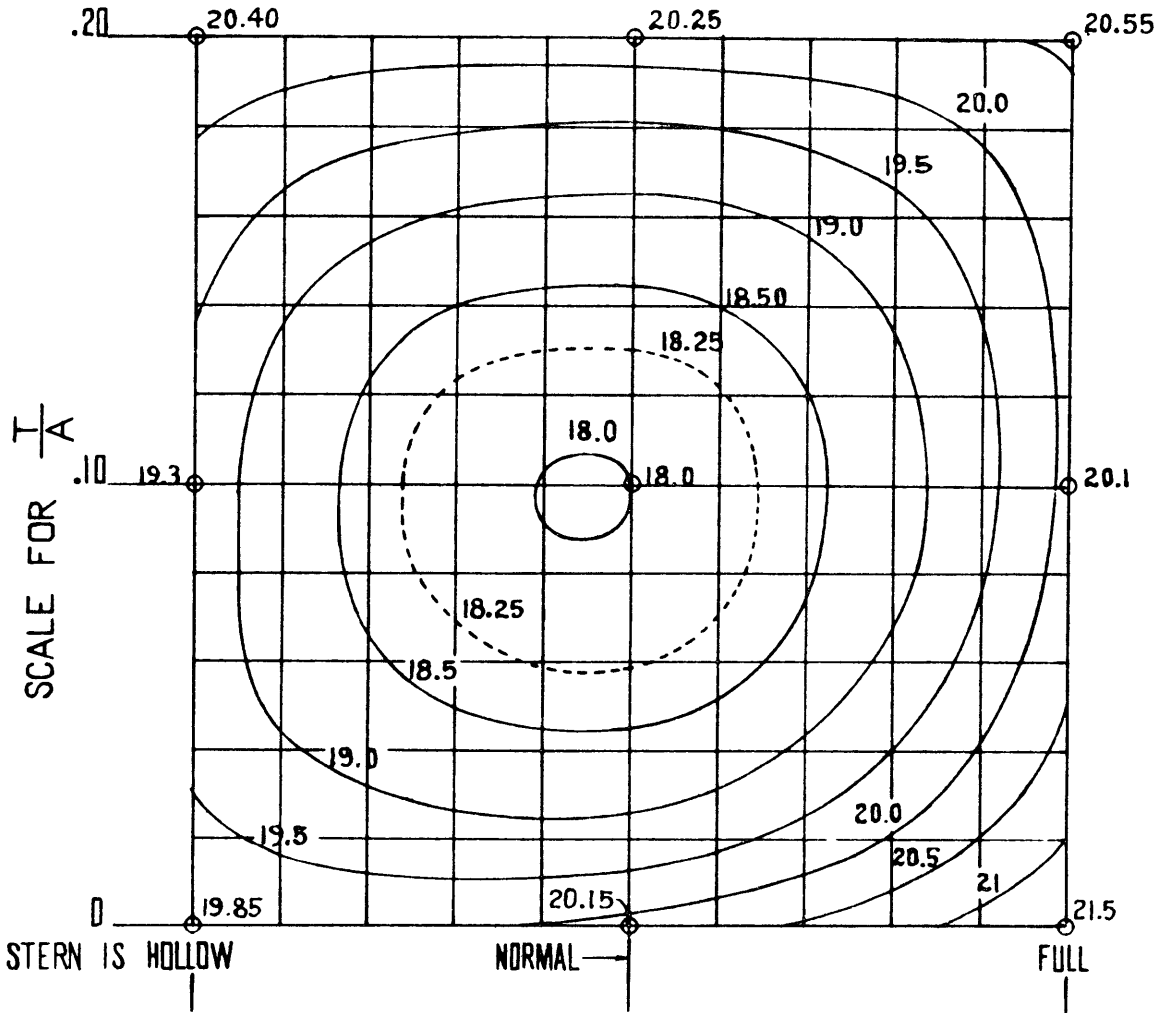
A = AREA OF MIDSHIP SECTION

HOLLOW, NORMAL, FULL, STERNS

			RESISTANCE IN POUNDS MODELS AT DESIGNED L.W.L.	
MODEL No.	CHARACTER OF STERN	$\frac{T}{A}$	V = 2.94 KNOTS (12 KN.)	V = 3.91 KNOTS (16 KN.)
3141	HOLLOW	0	3.55	6.70
3142	NORMAL	"	3.48	6.67
3143	FULL	"	3.80	6.81
3144	HOLLOW	.10	3.73	6.90
3145	NORMAL	"	3.91	6.96
3146	FULL	"	3.68	7.29
3147	HOLLOW	.20	4.11	7.42
3148	NORMAL	"	4.07	7.45
3149	FULL	"	3.78	7.22

FIG. 5a

CONTOURS OF TOTAL RESISTANCE, LBS.

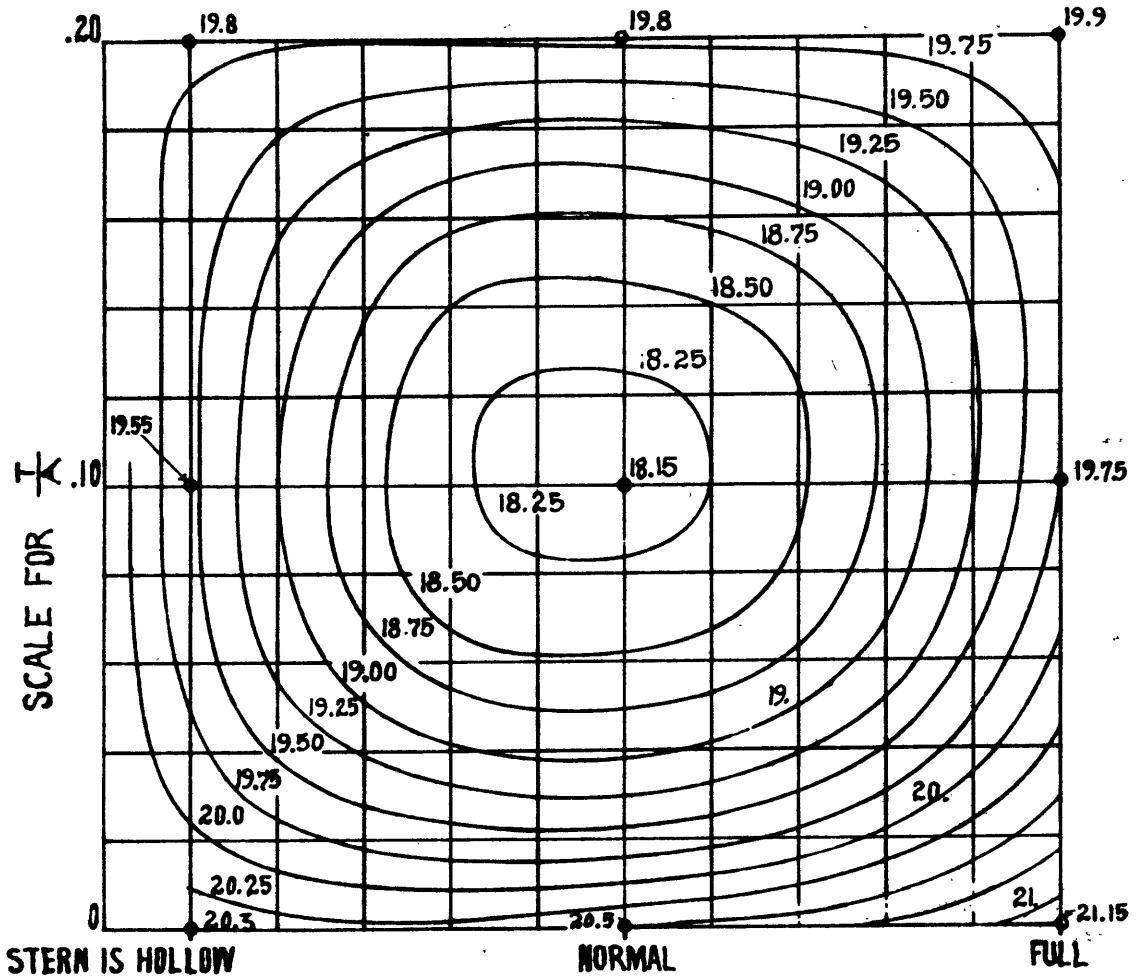


V = 6.0 KNOTS

DESIGNED W. L.

FIG. 6

CONTOURS OF TOTAL RESISTANCE, LBS.

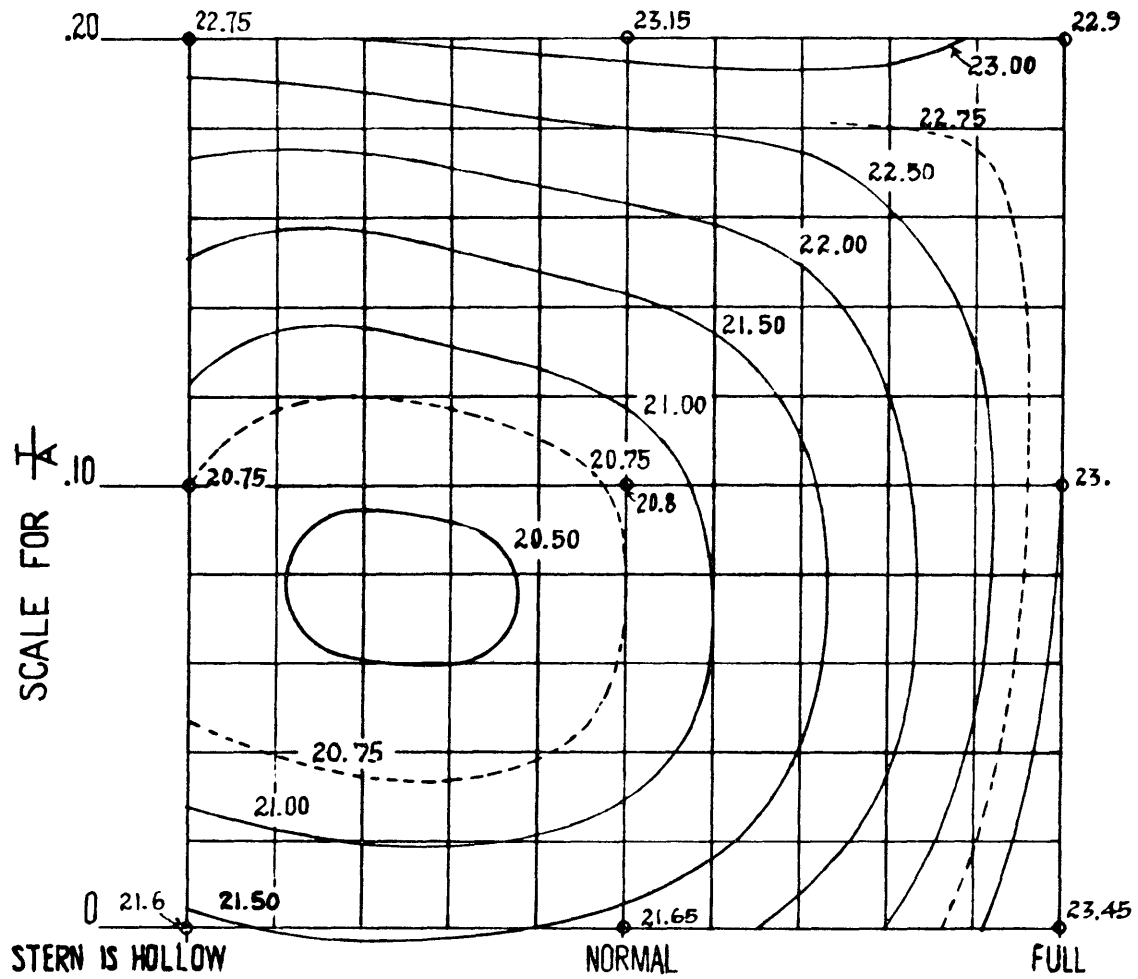


V = 6.0 KNOTS

3" BY BOW

FIG. 7

## CONTOURS OF TOTAL RESISTANCE, LBS.



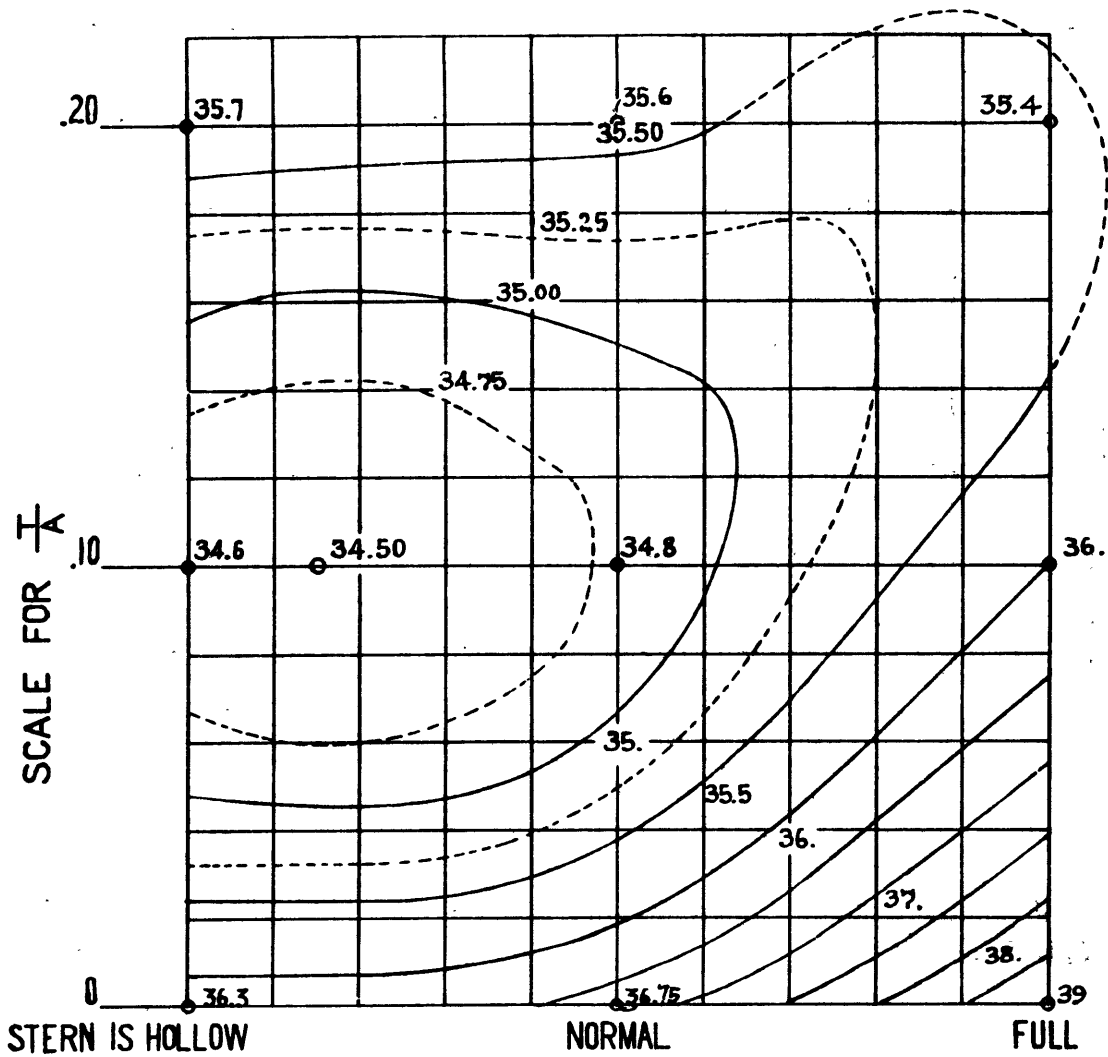
V = 6.0 KNOTS

3" BY STERN

FIG. 8



## CONTOURS OF TOTAL RESISTANCE, LBS.

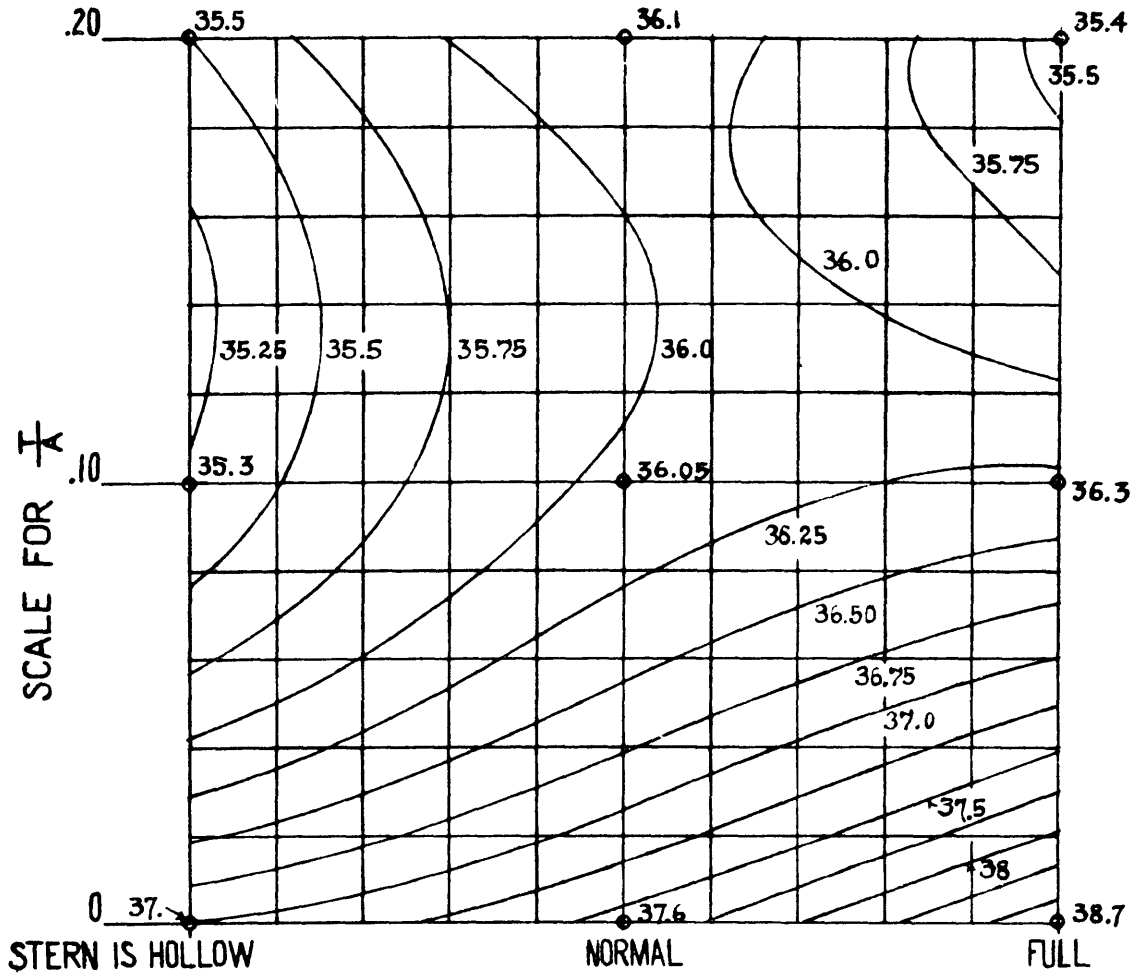


V = 7.5 KNOTS

DESIGNED W.L.

FIG. 9

CONTOURS OF TOTAL RESISTANCE, LBS.

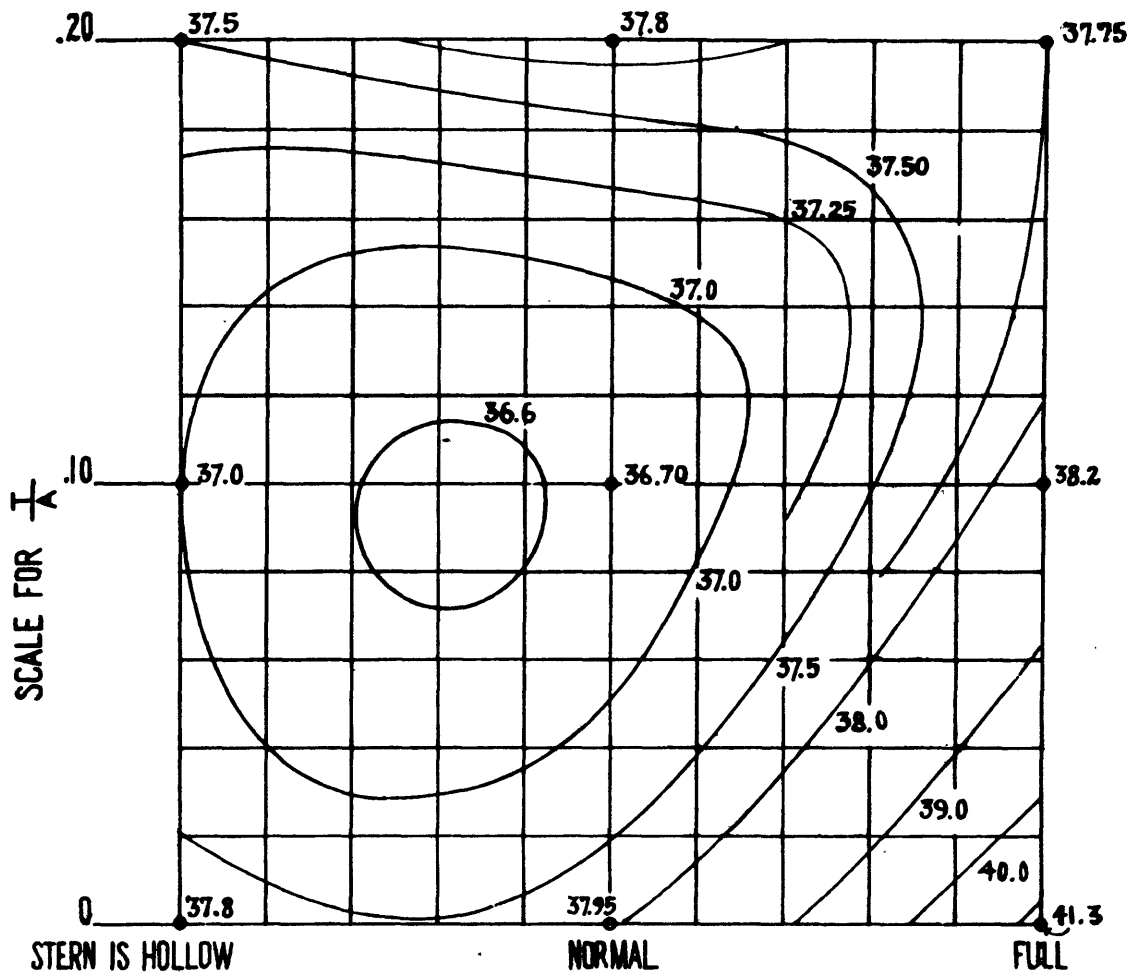


V = 7.5 KNOTS

3" BY BOW

FIG. 10

CONTOURS OF TOTAL RESISTANCE, LBS.

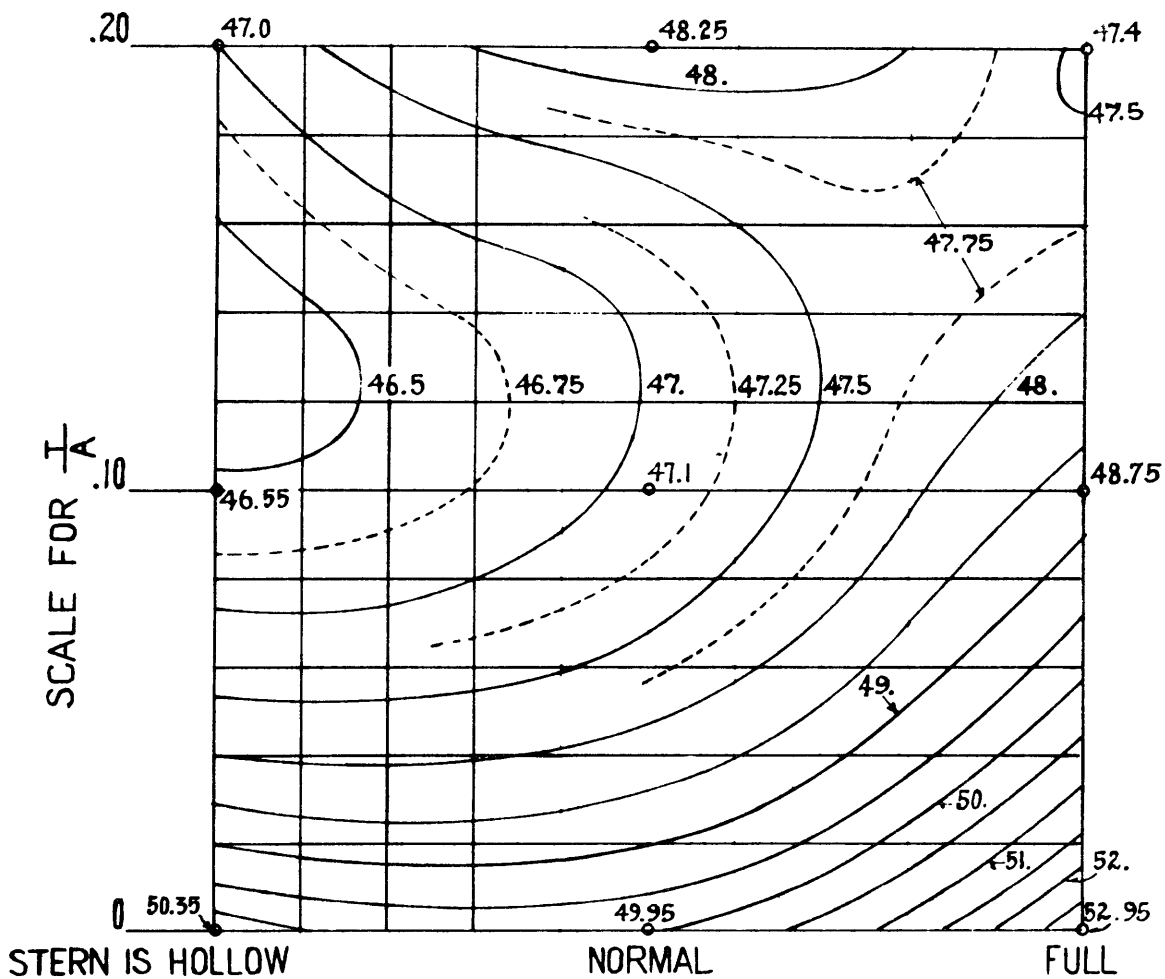


V = 7.5 KNOTS

3" BY STERN

FIG. 11

## CONTOURS OF TOTAL RESISTANCE, LBS.

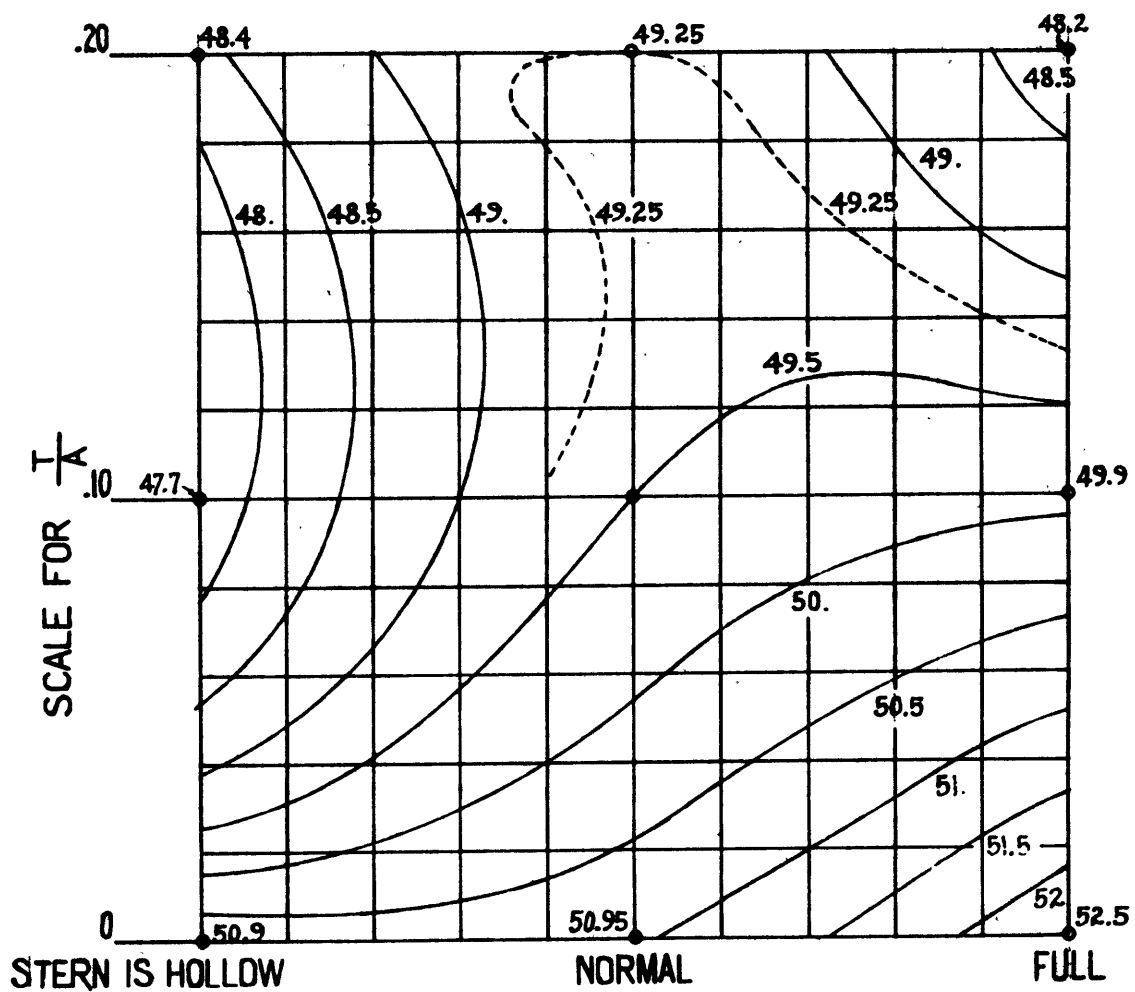


$V = 9.0$  KNOTS

DESIGNED W. L.

FIG. 12

## CONTOURS OF TOTAL RESISTANCE, LBS.

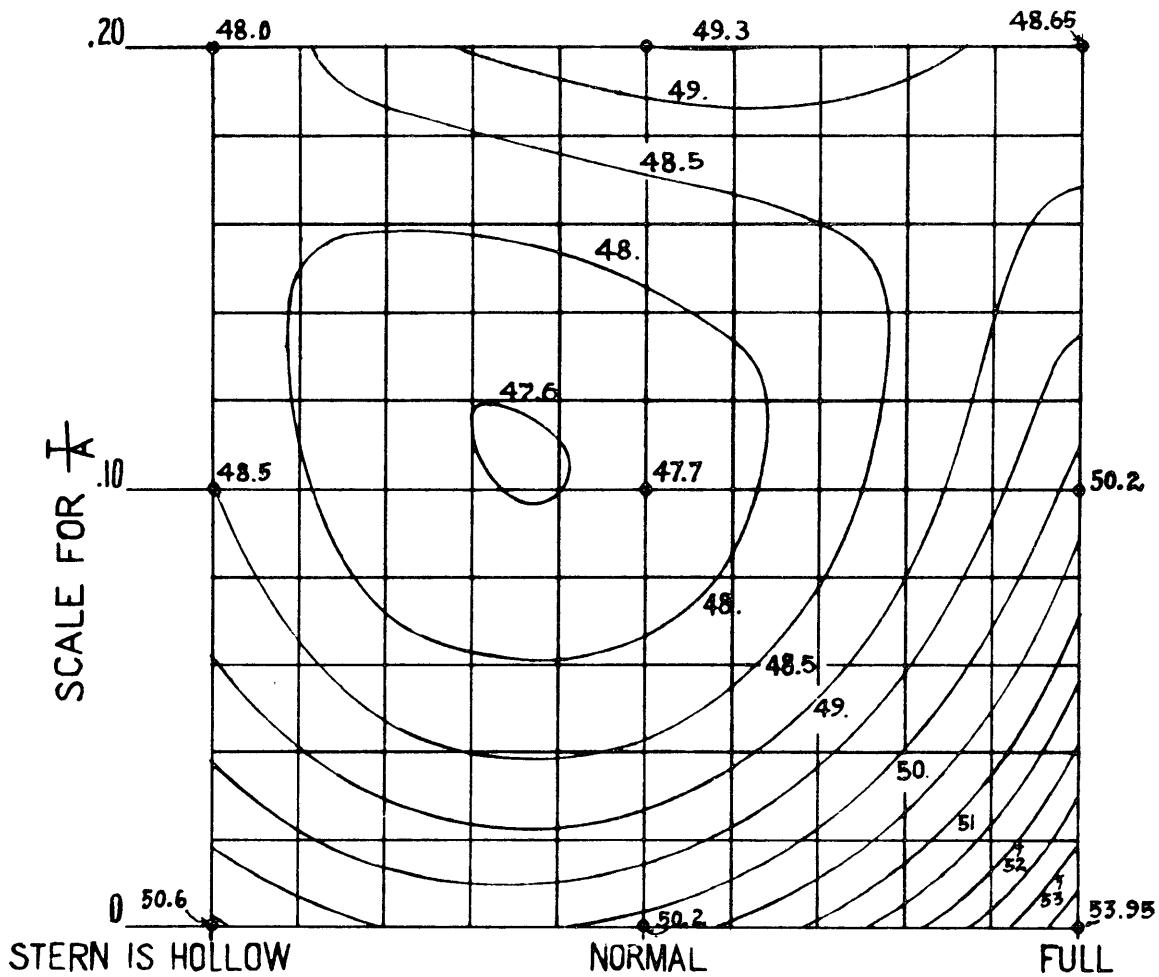


V = 9.0 KNOTS

3" BY BOW

FIG.13

CONTOURS OF TOTAL RESISTANCE, LBS.



V = 9.0 KNOTS

3" BY STERN

FIG. 14

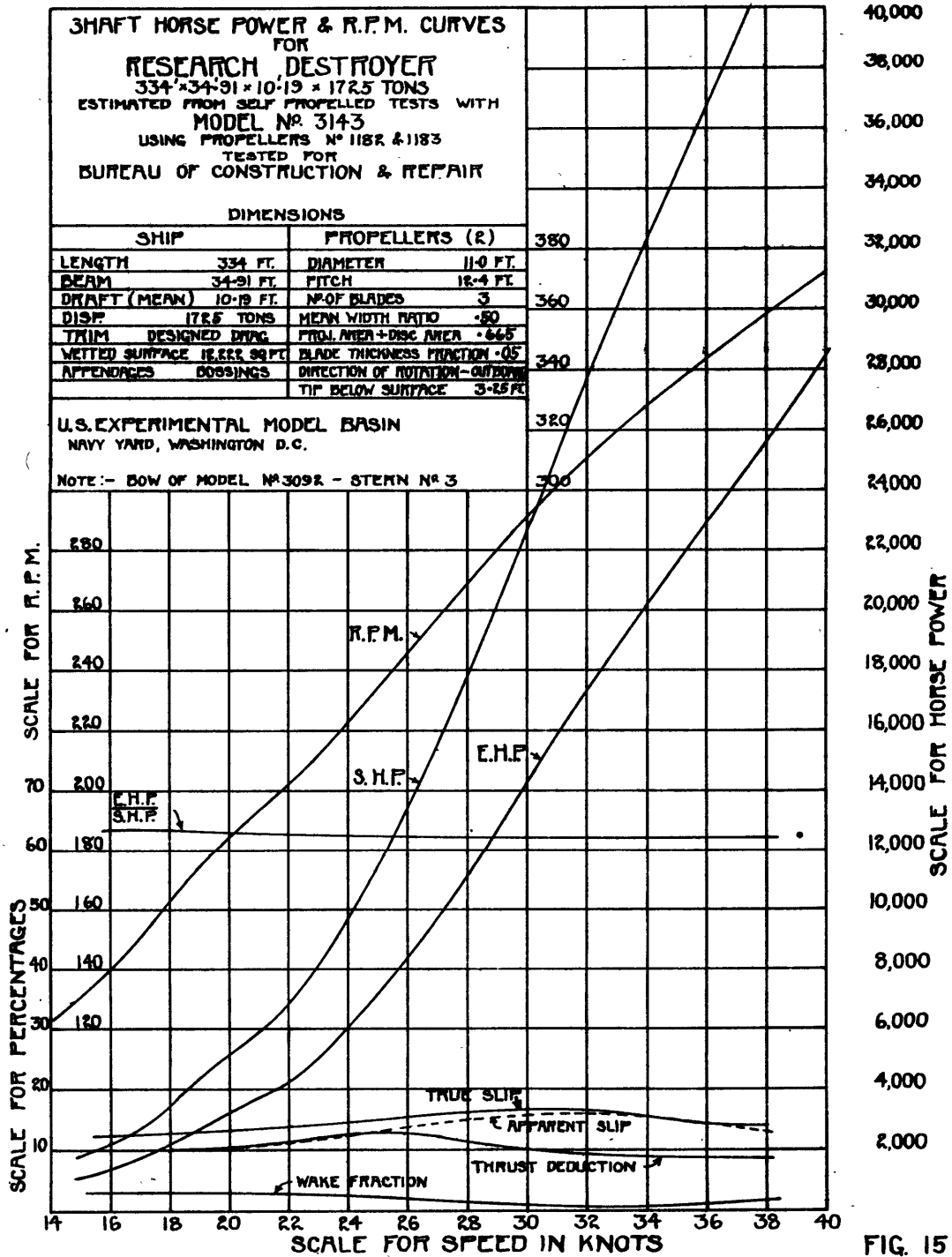


FIG. 15

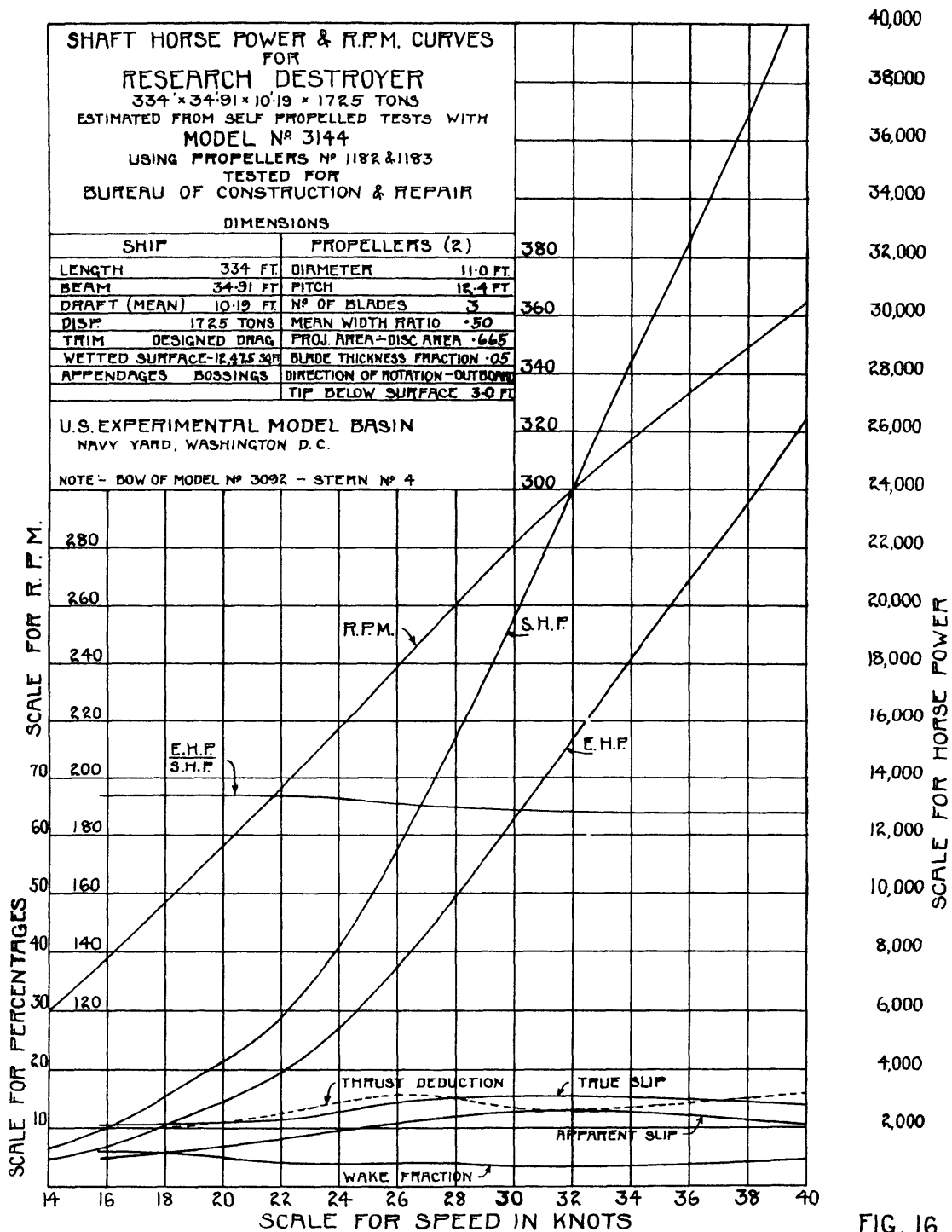


FIG. 16







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