

T2
4
5



V393
.R468

#2

NAVY DEPARTMENT
THE DAVID W. TAYLOR MODEL BASIN
WASHINGTON 7, D.C.

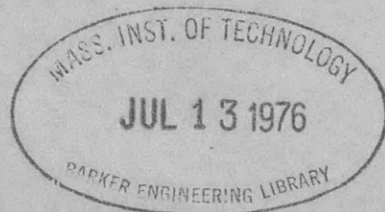
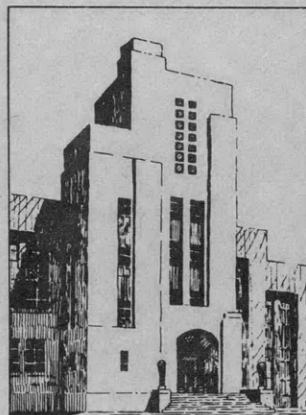
THE INFLUENCE OF THE BOUNDARY LAYER ON THE
WAVE RESISTANCE OF A SHIP

VLIIANIE POGRANICHNOGO SLOIA NA
VOLNOVOE SOPROTIVLENE KORABLIA

by

V.M. Lavrentieff

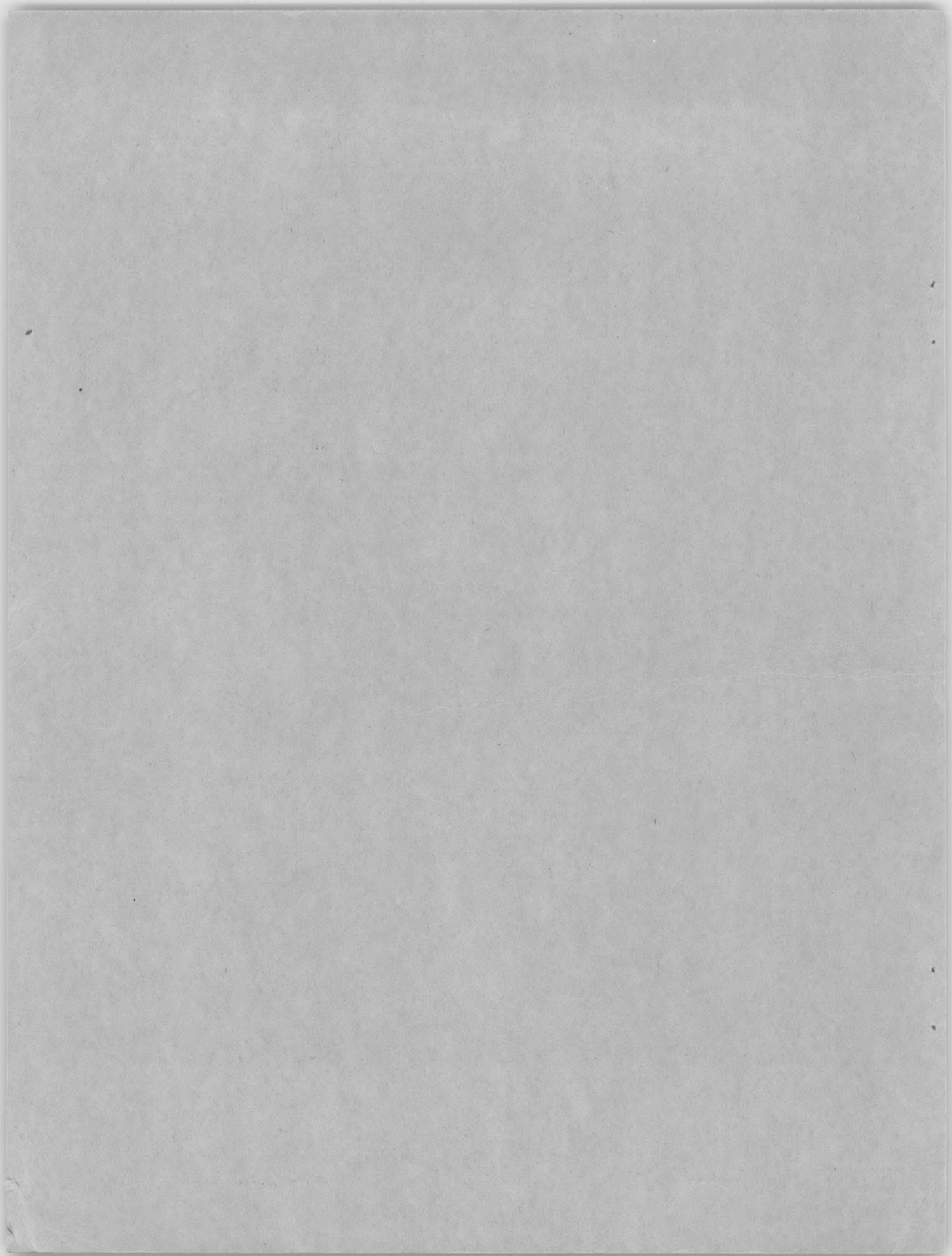
Academy of Sciences of the U.S.S.R.



Translated by Ralph D. Cooper

April 1952

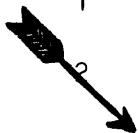
Translation 245



INITIAL DISTRIBUTION

Copies

- 10 Chief, BuShips, Project Records (Code 324), for distribution:
 - 5 Project Records
 - 1 Research (Code 300)
 - 1 Applied Science (Code 370)
 - 1 Ship Design (Code 410)
 - 1 Preliminary Design (Code 420)
 - 1 Technical Assistant to Chief of the Bureau (Code 106)
- 3 Chief, BuOrd, Underwater Ordnance (Re6a)
- 3 Chief, BuAer, Aerodynamics and Hydrodynamics Branch (DE-3)
- 2 Chief of Naval Operations, Op322F2
- 4 Chief of Naval Research, for distribution:
 - 3 Fluid Mechanics (N438)
 - 1 Undersea Warfare (466)
- 2 Commander, U.S. Naval Ordnance Laboratory, Silver Spring, Md.
- 2 Commander, Naval Ordnance Test Station, Inyokern, China Lake, Calif.
- 1 Commanding Officer and Director, U.S. Navy Underwater Sound Laboratory, Fort Trumbull, New London, Conn.
- 1 Commanding Officer, U.S. Naval Underwater Ordnance Station, Design Section, Newport, R.I.
- 2 Director, U.S. Naval Research Laboratory, Library Code 2021, Washington 20, D.C.
- 1 Director, Marine Physical Laboratory, U.S. Navy Electronics Laboratory, San Diego 52, Calif.
- 3 Director of Aeronautical Research, National Advisory Committee for Aeronautics, 1724 F Street, N.W., Washington, D.C.
- 2 Chief, National Hydraulic Laboratory, National Bureau of Standards, Washington 25, D.C., 1 for Dr. G.B. Schubauer
- 2 Director, Central Air Documents Office, U.B. Building, 4th and Main Streets, Dayton 2, Ohio
- 2 Director, Experimental Towing Tank, Stevens Institute of Technology, 711 Hudson Street, Hoboken, N.J.
- 2 Newport News Shipbuilding and Dry Dock Co., Newport News, Va., for distribution:
 - 1 Senior Naval Architect
 - 1 Supervisor, Hydraulics Laboratory
- 1 Director, Iowa Institute of Hydraulic Research, State University of Iowa, Iowa City, Iowa
- 1 Director, Hydrodynamics Laboratories, California Institute of Technology, Pasadena 4, Calif.
- 2 Director, Experimental Naval Tank, Department of Naval Architecture and Marine Engineering, University of Michigan, Ann Arbor, Mich.
- 2 Head, Department of Naval Architecture and Marine Engineering, Massachusetts Institute of Technology, Cambridge 39, Mass., 1 for Prof. M. Abkowitz
- 1 Dean, School of Engineering, University of Notre Dame, Notre Dame, Ind.



Copies

- 1 Administrator, Webb Institute of Naval Architecture, Crescent Beach Road, Glen Cove, Long Island, N.Y.
- 1 Director, Institute for Fluid Dynamics and Applied Mathematics, University of Maryland, College Park, Md.
- 1 Director, Fluid Mechanics Laboratory, New York University, New York 53, N.Y.
- 1 Director, Fluid Mechanics Laboratory, Columbia University, New York 27, N.Y.
- 1 Director, Scripps Institution of Oceanography, University of California, La Jolla, Calif.
- 1 Director, Woods Hole Oceanographic Institution, Woods Hole, Mass.
- 2 Director, Applied Physics Laboratory, Johns Hopkins University, 8621 Georgia Avenue, Silver Spring, Md., 1 for Dr. F.N. Frenkiel
- 1 Supt. U.S. Naval Postgraduate School, Monterey, Calif.
- 1 Editor, Bibliography of Technical Reports, Office of Technical Services, U.S. Department of Commerce, Washington 25, D.C.
- 1 Editor, Engineering Index, 29 West 39th Street, New York 18, N.Y.
- 1 Librarian, American Society of Mechanical Engineers, 29 West 39th Street, New York 18, N.Y.
- 1 Librarian, American Society of Civil Engineers, 33 West 39th Street, New York 18, N.Y.
- 1 Librarian, Daniel Guggenheim Aeronautical Laboratory, California Institute of Technology, Pasadena 4, Calif.
- 1 Librarian, Case Institute of Technology, Cleveland 6, Ohio
- 1 Librarian, Franklin Institute, Parkway at 20th Street, Philadelphia, Pa.
- 1 Librarian, Institute of the Aeronautical Sciences, 2 East 64th St., New York 21, N.Y.
- 1 Librarian, Aeronautical Engineering Library, Massachusetts Institute of Technology, Cambridge 39, Mass.
- 1 Librarian, Pacific Aeronautical Library, Institute of the Aeronautical Sciences, 7660 Beverly Blvd., Los Angeles 36, Calif.
- 1 Prof. A.D. Hay, Princeton University, Princeton, N.J.
- 1 Dr. G.F. Wislicenus, Mechanical Engineering Department, Johns Hopkins University, Baltimore 18, Md.
- 1 Dr. V.L. Streeter, Director, Fundamental Fluid Research, Illinois Institute of Technology, Technology Center, Chicago 16, Ill.
- 1 Dr. J.V. Wehausen, Editor, Mathematical Reviews, Providence, R.I.
- 1 Mr. J.P. Breslin, Gibbs and Cox, Inc., 21 West Street, New York 6, N.Y.
- 1 Prof. J.J. Stoker, Institute for Mathematics and Mechanics, New York University, New York 53, N.Y.
- 1 Prof. Fritz John, Institute for Mathematics and Mechanics, New York University, New York 53, N.Y.
- 1 Dr. C.J. Peirce, Headquarters USAFE, APO 633, c/o P.M., New York, N.Y.

Copies

- 9 British Joint Services Mission (Navy Staff) P.O. Box 165, Benjamin Franklin Station, Washington, D.C.
- 2 Australian Scientific Research Liaison Office, Washington, D.C.
- 3 Hydrodynamics Laboratory, National Research Council, Ottawa, Canada
- 1 Capt. R. Brard, Directeur du Bassin d'Essais des Carènes, 6 Boulevard Victor, Paris (15e), France
- 1 Dr. L. Malavard, Office National d'Etudes et de Recherches Aéronautique, 25 Avenue de la Division-LeClerc, Chatillon, Paris, France
- 1 Gen. Ing. U. Pugliese, Presidente, Istituto Nazionale per Studi ed Esperienze di Architettura Navale, Via della Vasca Navale 89, Rome, Italy
- 1 Sr. M. Acevedo y Campoamor, Director, Canal de Experiencias Hidrodinámicas, El Pardo, Madrid, Spain
- 1 Dr. J. Dieudonné, Directeur, Institut de Recherches de la Construction Navale, 1 Boulevard Haussmann, Paris (9e), France
- 1 Prof. L. Troost, Superintendent, Nederlandsh Scheepsbouwkundig Proefstation, Haagsteeg 2, Wageningen, The Netherlands
- 1 Prof. J.K. Lunde, Skipsmodelltanken, Tyholt Trondheim, Norway
- 1 Prof. H. Nordstrom, Director, Statens Skeppsprovvningsanstalt, Göteborg 24, Sweden
- 1 Dr. S.L. Smith, Director, British Shipbuilding Research Association, 5 Chesterfield Gardens, Curzon Street, London, W. 1, England
- 1 Dr. J.F. Allan, Superintendent, Ship Division, National Physical Laboratory, Teddington, Middlesex, England
- 1 Dr. Jun-ichi Okabe, The Research Institute for Applied Mechanics, Kyushu University, Hakozaki-machi Fukuoka-shi, Japan

THE INFLUENCE OF THE BOUNDARY LAYER ON THE
WAVE RESISTANCE OF A SHIP

(Vliianie Pogranichnogo Sloia na Volnovoe Soprotivlenie Korablia)

by

V.M. Lavrentieff

Central Scientific Research Institute of the Maritime Fleet

Reports of the Academy of Sciences of the U.S.S.R.
1951, Vol. LXXX, No. 6
(Presented by Academician A.I. Nekrasov, 20 August 1951)

Translated by Ralph D. Cooper

April 1952

Translation 245

THE INFLUENCE OF THE BOUNDARY LAYER ON THE
WAVE RESISTANCE OF A SHIP

The experimental investigation of the linear theory of the wave resistance of a ship in an ideal fluid shows that, in general, this theory is satisfactorily confirmed by tests for sufficiently sharp closed bodies with the exception of the range of gravitational numbers (Froude numbers), which correspond to the interfering humps and hollows on the resistance curves. It is known that the viscosity of the fluid considerably smooths out the modulation in wave resistance curves.

The influence of the viscosity of the fluid on the wave resistance is manifested mainly through the influence of the boundary layer. This last can be taken into account approximately, if, in place of flow about the actual ship with wetted surface $y = \phi(x, z)$, the flow of an ideal fluid about a body (Figure 1) with wetted surface $y = f(x, z) = \phi(x, z) + \delta^*(x, z)$ is considered, which represents the hull of the ship surrounded by a displacement layer δ^* together with its wake, whose width may be approximated by a constant equal to twice the thickness δ^* at the stern perpendicular. Such an assumption is equivalent to considering that separation of the boundary layer does not occur, a condition which is satisfied practically by all vessels for which a study of wave resistance has any significance.

The thickness of the displacement layer δ^* can be determined approximately at every point on the surface of a ship by the method of successive approximations; however, it should be stated that for practical purposes it will be sufficient to use the available solution for a flat plate having the same length and draft as the ship.

For determining wave resistance in deep water (the formulas for shallow water, either in depth or in breadth, are obtained in an analogous way) we use the well-known formulas:

$$R_w = \frac{4\gamma g}{\pi v^2} \int_0^{\pi/2} (I^2 + J^2) \sec^3 \theta d\theta \quad [1]$$

$$\left. \begin{matrix} I(\theta) \\ J(\theta) \end{matrix} \right\} = \int_S \int \frac{\partial f}{\partial x} \exp \left(\frac{gz}{v^2} \sec^2 \theta \right) \left\{ \begin{matrix} \cos \left(\frac{gx}{v^2} \sec \theta \right) \\ \sin \left(\frac{gx}{v^2} \sec \theta \right) \end{matrix} \right\} dx dz \quad [2]$$

where γ is the specific weight of water,
 g is the acceleration of the force of gravity,
 v is the speed of the ship, and
 $y = f(x, z)$ is the equation of the surface of the body.

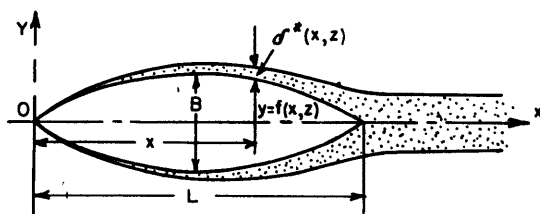


Figure 1

In the integrals [2], the integration extends over the center-line area S of the ship, the axis OZ being directed upward.

Designating the length, beam, and draft of the ship by L , B , and T , we introduce the nondimensional quantities:

$$\xi = \frac{x}{L}; \eta = \frac{y}{B/2}; \zeta = \frac{z}{T}; f(x, z) = \frac{B}{2} f(\xi, \zeta); \frac{\partial f}{\partial x} = \frac{1}{4} \frac{B}{L} \frac{\partial f}{\partial \xi} \quad [3]$$

$F = v/\sqrt{gL}$ is the gravitational number.

Then Formulas [1] and [2] take the form

$$R_w = \frac{\gamma}{\pi} \frac{B^2 T^2}{L F^2} \int_0^{\pi/2} (i^2 + j^2) \sec^3 \theta d\theta \quad [1a]$$

$$\left. \begin{matrix} i(\theta) \\ j(\theta) \end{matrix} \right\} = \int_S \frac{\partial f}{\partial \xi} \exp\left(\frac{T}{L} \frac{\sec^2 \theta}{F^2} \zeta\right) \left\{ \begin{matrix} \cos\left(\frac{\sec \theta}{F^2} \xi\right) \\ \sin\left(\frac{\sec \theta}{F^2} \xi\right) \end{matrix} \right\} d\xi d\zeta \quad [2a]$$

The wave resistance per ton of displacement will be

$$\frac{R_w}{\Delta} = \frac{1}{\pi \delta} \frac{B}{L} \frac{T}{L} \frac{1}{F^2} \int_0^{\pi/2} (i^2 + j^2) \sec^3 \theta d\theta \quad [4]$$

where $\Delta = \gamma \delta B L T$ is the displacement of the ship and δ is the block coefficient.

Keeping in mind that

$$f(\xi, \zeta) = \phi(\xi, \zeta) + \delta^*(\xi, \zeta) \quad [5]$$

we find for the resistance per ton of displacement, the formula*

$$\begin{aligned} \frac{R_w}{\Delta} = & \frac{1}{\pi \delta} \frac{B}{L} \frac{T}{L} \frac{1}{F^2} \left\{ \int_0^{\pi/2} (i_\infty^2 + j_\infty^2) \sec^3 \theta d\theta + \right. \\ & \left. + 2 \int_0^{\pi/2} (i_\infty i_1 + j_\infty j_1) \sec^3 \theta d\theta + \int_0^{\pi/2} (i_1^2 + j_1^2) \sec^3 \theta d\theta \right\} \quad [6] \end{aligned}$$

*Translator's Note: In the original paper, because of a typographical error, the second integrand was incorrectly given as $\{(i_\infty i_1 + j_\infty j_1) \sec^3 \theta d\theta\}$

where:

$$\left. \begin{matrix} i_{\infty}(\theta) \\ j_{\infty}(\theta) \end{matrix} \right\} = \iint_S \frac{\partial \phi}{\partial \xi} \exp\left(\frac{T}{L} \frac{\sec^2 \theta}{F^2} \xi\right) \left\{ \begin{matrix} \cos\left(\frac{\sec \theta}{F^2} \xi\right) \\ \sin\left(\frac{\sec \theta}{F^2} \xi\right) \end{matrix} \right\} d\xi d\zeta \quad [7]$$

$$\left. \begin{matrix} i_1(\theta) \\ j_1(\theta) \end{matrix} \right\} = \iint_S \frac{\partial \delta^*}{\partial \xi} \exp\left(\frac{T}{L} \frac{\sec^2 \theta}{F^2} \xi\right) \left\{ \begin{matrix} \cos\left(\frac{\sec \theta}{F^2} \xi\right) \\ \sin\left(\frac{\sec \theta}{F^2} \xi\right) \end{matrix} \right\} d\xi d\zeta \quad [8]$$

From Equation [6] it follows that the wave resistance of a ship in a viscous fluid can be represented as consisting of three parts:

1. The resistance of the ship in an ideal fluid (at a Reynolds number $Re = vL/\nu = \infty$), corresponding to the first term of [6];
2. the wave resistance of a flat plate caused by the presence of its boundary layer (the third term in [6]), and
3. the wave resistance obtained as a result of the interference of the wave motions created separately by the hull of the ship and by the boundary layer, to which the second term of [6] corresponds.

At higher values of the Reynolds number the third term of [6] proves to be of second order of smallness in comparison to the second and may be neglected.

The most simple way to make computations with the formulas is to start from the exponential expressions for the boundary layer. Let us assume, for instance, "the one-seventh power law," taking into account that at the aft end in the region of positive pressure gradient, the velocity profile in the boundary layer is less full than for a flat plate (as is known, for a flat plate in the region of usual Reynolds numbers, "the one-eleventh power law" gives good agreement with experiments).

Thus, we shall start with the formula

$$\delta^*(\xi) = \frac{\delta^*(x)}{B/2} = 0.0924 \frac{L}{B} \frac{\xi^{4/5}}{\sqrt[5]{Re}} \quad [9]$$

In this case Formula [8] takes the form:

$$\left. \begin{matrix} i_1(\theta) \\ j_1(\theta) \end{matrix} \right\} = 0.074 \frac{L}{B} \frac{1}{\sqrt[5]{Re}} \iint_S \frac{\exp\left(\frac{T}{L} \frac{\sec^2 \theta}{F^2} \xi\right)}{\sqrt[5]{\xi}} \left\{ \begin{matrix} \cos\left(\frac{\sec \theta}{F^2} \xi\right) \\ \sin\left(\frac{\sec \theta}{F^2} \xi\right) \end{matrix} \right\} d\xi d\zeta \quad [8a]$$

Since, in our assumption, the displacement thickness δ^* appears as a function of ξ only and does not depend on ζ , and if, in addition, for the purpose of calculating i_1 and j_1 we consider the centerline area of the ship as a rectangle, the integration over ζ can be carried out beforehand, giving:

$$\left. \begin{matrix} i_1(\theta) \\ j_1(\theta) \end{matrix} \right\} = 0.074 \frac{L}{B} \frac{L}{T} \frac{F^2 \cos^2 \theta}{\sqrt{Re}} \left[\exp\left(-\frac{T}{L} \frac{\sec^2 \theta}{F^2}\right) - 1 \right] \int_0^1 \xi^{-1/5} \left\{ \begin{matrix} \cos\left(\frac{\sec \theta}{F^2} \xi\right) \\ \sin\left(\frac{\sec \theta}{F^2} \xi\right) \end{matrix} \right\} d\xi \quad [8b]$$

From this it is seen that if the wave resistance of the plate is of order $Re^{-2/5}$, then the wave resistance created by the interference (the second term in [6]) will be of order $Re^{-1/5}$.

Analogous formulas for i_1 and j_1 can be obtained by assuming an arbitrary exponential law for the velocity distribution in the boundary layer. In particular, for a laminar layer the integrals for i_1 and j_1 are expressed in final form by Fresnel integrals.

The practical calculation of the integrals for i_1 and j_1 presents no difficulties, and the calculation of wave resistance including the effect of the boundary layer can be made for closed bodies, which are given either analytically or in the form of ordinates of the theoretical plan. In the latter case, it is necessary to use the transformation of G.E. Pavlenko.

Formulas [6] and [8a] show that the wave resistance has a "measurable effect."

Submitted 18 June 1951

MIT LIBRARIES

DUPL



3 9080 02993 0044

